

North America

Suggested Torque Values for Use with Teadit Spiral wound 913M (with inner & outer rings)/Standard ASME B16.5 Raised Face Pipe Flanges

## For Use with ASTM A193 B7 Bolting or equal yield strength bolt material.

Gasket to be per ASME B16.20 and used with appropriate flange bore size per the B16.20 tables. If no inner ring is used, gasket may buckle on the ID. Use Correct size, New, bolts/studs, nuts, washers (multiply torques by 0.70 for PTFE coated bolting). Lubricate on bolt threads and nut faces with compatible antiseize\*. This Chart gives the torque value for the final pass. After hand tightening, torquing must follow a cross bolting sequence as exampled in Annex 12.1 of Teadit's "Industrial Gaskets" 3rd editon by Jose Veiga. There shall be 3 complete passes (30%, 60%,100% of final pass torque). Once final torque is achieved, a minimum of 2 clockwise passes to be applied until there is no further nut rotation.

Class 150

Nominal Pipe Size Torque FT. LB. 1/2 40 3/4 60 60 1 1/4 60 1 1/2 60 120 2 1/2 120 125 3 1/2 120 115 200 200 225 10 320 12 320 14 500 16 405 18 650 20 595 24 835

**Class 300** 

Nominal Pipe Size	Torque FT. LB.
1/2	40
3/4	65
1	90
1 1/4	105
1 1/2	170
2	90
2 1/2	115
3	160
3 1/2	200
4	200
5	200
6	200
8	320
10	500
12	710
14	535
16	835
18	835
20	835
24	1200

Flange stress limitations were considered per Warren Brown and David Reeves, <u>An Update on Selecting the Optimum Bolt Assembly Stress for Piping Flanges</u>, (Advanced Draft for presentation at 2007 ASME PVP Conference), Table 2. This assumes A-105 or stainless steel weld necks w/ pipe walls as listed by the document. Other arrangements may require further evaluation. Spiral wound gasket dimensions were used for the flange stress analysis. Not suitable for flange materials with elongation at failure less than 20%.

<sup>\*</sup>Charts based on nut factor approximately 0.17



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<u>Class 400</u> <u>Class 600</u>

Nominal Pipe Size	Torque FT. LB.
1/2	No flanges
3/4	No flanges
1	No flanges
1 1/4	No flanges
1 1/2	No flanges
2	No flanges
2 1/2	No flanges
3	No flanges
3 1/2	No flanges
4	320
5	320
6	320
8	500
10	620
12	875
14	875
16	1200
18	1200
20	1400
24	2600

Nominal Pipe Size	Torque FT. LB.
1/2	40
3/4	60
1	85
1 1/4	85
1 1/2	160
2	85
2 1/2	160
3	180
3 1/2	300
4	330
5	470
6	470
8	650
10	875
12	875
14	1020
16	1335
18	1900
20	1900
24	3000

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<u>Class 900</u> <u>Class 1500</u>

Nominal Pipe Size	Torque FT. LB.
1/2	No Flanges
3/4	No Flanges
1	No Flanges
1 1/4	No Flanges
-	
1 1/2	No Flanges
2	No Flanges
2 1/2	No Flanges
3	265
4	500
5	840
6	590
8	950
10	950
12	1130
14	1330
16	1830
18	3000
20	3000
24	5000

Nominal Pipe Size	Torque FT. LB.
1/2	80
3/4	100
1	160
1 1/4	200
1 1/2	275
2	200
2 1/2	300
3	400
4	650
5	1000
6	900
8	1400
10	2400
12	2500
14	3200
16	4500
18	6000
20	7730
24	12750

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# **Class 2500**

Nominal Pipe Size	Torque FT. LB.
1/2	85
3/4	85
1	125
1 1/4	220
1 1/2	320
2	220
2 1/2	320
3	450
4	750
5	1300
6	2000
8	2000
10	3500
12	5000

<sup>\*</sup>Chart based on nut factor approximately 0.17

Flange stress limitations were considered per Warren Brown and David Reeves, An Update on Selecting the Optimum Bolt Assembly Stress for Piping Flanges, (Advanced Draft for presentation at 2007 ASME PVP Conference), Table 2. This assumes A-105 or stainless steel weld necks with pipe walls as listed by the document. Other arrangements may require further evaluation. Spiral wound gasket dimensions were used for the flange stress analysis. Not suitable for flange materials with elongation at failure less than 20%.